

# Parametric Design and Optimization of Multi-Rotor Aerial Vehicles

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3**C. Ampatis and E. Papadopoulos**

**Abstract** This work addresses the problem of optimal selection of propulsion components for a multi-rotor aerial vehicle (MRAV), for a given payload, payload capacity, number of rotors, and flight duration. Considering that the main components include motors, propellers, electronic speed controllers (ESC), and batteries, a steady state model is developed for each component using simplified analysis. Based on technical specifications of commercially available batteries, motors and ESCs, component functional parameters identified earlier were expressed as a function of component size, in terms of an equivalent length. Propeller models were developed using available experimental data. Airframe dimensions and total weight were expressed as a function of propeller diameter, number of rotors, and maximum thrust. Using Matlab's "fmincon" function, a program was developed which calculates the optimal design vector using the total energy consumption and vehicle diameter as objective function. Using the developed program, the influence of the payload and of the number of rotors on the design vector and the MRAV size was studied. The results obtained by the program were compared to existing commercial MRAVs.

**Keywords** Multi-rotor aerial vehicle (MRAV) design • Parametric design • Constrained optimization • Energy and size minimization

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N.J. Daras (ed.), *Applications of Mathematics and Informatics in Science and Engineering*,  
Springer Optimization and Its Applications 91, DOI 10.1007/978-3-319-04720-1\_1,  
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## Introduction

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Recently, Multi-Rotor Aerial Vehicles (MRV) are encountered in an increasing number of military and civilian applications. A particular advantage an MRV has over other aerial vehicles is its unique ability for vertical stationary flight (VTOL). Micro and mini MRVs with payload capabilities of up to 100 g and 2 kg respectively [1] offer major advantages when used for aerial surveillance and inspection in complex and dangerous indoor and outdoor environments. In addition, improvements and availability in cost-effective batteries and other technologies are rapidly increasing the scope for commercial opportunities.

In most MRV configurations, rotors are in the same plane and symmetrically fixed on the airframe. The number of rotors is always even in order to balance the torque produced by the rotors. An exception is the trirotor, where one rotor is placed on a tilting mechanism in order to balance the excess torque. Additional configurations include MRVs with multiple pairs of coaxial-counter rotating rotors. However, researchers push the limits by studying different configurations where the rotors are not in the same plane but placed arbitrarily in 3D space [2], or even having the ability of thrust vectoring [3, 4].

In any configuration, an MRV design consists of basic components, such as batteries, electric motors, and propellers, which constitute the vehicle propulsion system. One of the most critical stages in MRV design is the proper motor-propeller matching. The electric motor market offers a large range of motors for almost any application, thus an MRV designer does not need to design the motor. Propellers used for MRV applications are taken from the remote controlled (RC) aircraft market, therefore they are designed for RC aircrafts. However, an MRV hovers for a great percent of the total flight time, therefore needs propellers designed for maximum hover efficiency. Recently, the MRV industry produced such propellers but in a limited range. Recent studies resulted in optimized designs of micro and mini rotorcraft vehicle propellers that are easy to manufacture, such as curved plate plastic propellers, [5, 6].

Apart from optimizing each MRV component separately, an MRV designer would benefit from an automated design method that would take into account all design requirements to yield an optimized combination of commercially available components. Although studies on automated design methods exist [7, 8], no method exists that takes into account both the propulsion system modeling and the functional parameters of existing components.

In this paper, we propose an MRV design method, which selects the optimum propulsion system components. Given the MRV design requirements such as payload, payload capacity, number of rotors, and flight duration, a Matlab program calculates the propulsion system components and MRV size which leads to an energy-efficient design, or to a design with the smallest size. To achieve this we use simplified models for each component, and expressions of component functional parameters as a function of component size, using their commercially available technical specifications.

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**Component and System Modeling**

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The components to be modeled include the electric motors, the electronic speed controller, batteries, propellers, and the airframe. Combining the simplified models will lead to a system model for the MRAV steady state operation.

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***Electric Motor Model***

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The electric motors used in MRAV applications are outrunner Brushless Direct Current (BLDC) ones. This is due to their high efficiency and high torque constant ( $K_T$ ), which allows direct propeller coupling (no gearbox). Although a BLDC motor is a synchronous 3-phase permanent magnet motor, it can be modeled as a permanent magnet DC motor. This leads to a classic three-constant model, see Fig. 1.

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In Fig. 1,  $V_k$  is the supply voltage (V),  $i_a$  is the current through the motor coils (A),  $e_a$  is the back-electromotive force (EMF) (V),  $R_a$  is the armature resistance ( $\Omega$ ),  $M$  is the torque produced by the motor (Nm), and  $\omega$  is its shaft angular velocity (rad/s). The equations describing the motor are:

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$$V_k = e_a + i_a R_a \tag{1}$$

$$e_a = K_e \omega = K_T i_a = N / K_V \tag{2}$$

where  $K_e$  is the motor back EMF constant (Vs/rad),  $K_T$  is the motor torque constant (Nm/A),  $N$  is the motor rpm, and  $K_V$  is motor speed constant (rpm/V). The  $K_T$  is related to  $K_V$  by:

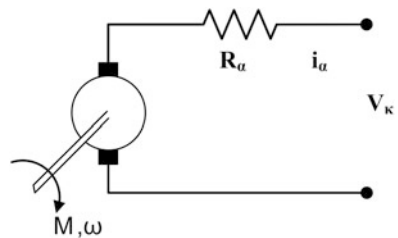
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$$K_e = K_T = \frac{30}{\pi} \frac{1}{K_V} \tag{3}$$

The total torque produced by the motor is:

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$$M = K_T i_a \tag{4}$$



**Fig. 1** Electric motor model

The output torque is:

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$$M_{\text{mot}} = K_T (i_a - i_0) \quad (5)$$

where  $i_0$  is the no-load current. The motor input power is:

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$$P_{\text{in}} = V_k i_a \quad (6)$$

the motor output power is:

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$$\begin{aligned} P_{\text{mot}} &= M_{\text{mot}} \omega = K_T (i_a - i_0) \omega = e_a (i_a - i_0) \\ &= (V_k - i_a R_a) (i_a - i_0) \end{aligned} \quad (7)$$

and the motor speed in rpm is:

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$$N = (V_k - i_a R_a) K_V \quad (8)$$

Given the parameters  $K_T$ ,  $R_a$ , and  $i_0$  we can calculate the performance of the motor.

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### ***Electronic Speed Controller Model***

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Electronic speed controllers regulate motor speed within a range depending on the load and battery voltage. The important quantity here is the ESC power losses, caused by its power MOSFETs. The major parameters that affect ESC power losses are the transistor drain-to-source "ON" state resistance  $R_{\text{DS(ON)}}$ , transistor characteristics on transient operation, and the frequency switching the transistor "ON" and "OFF." Power losses at full throttle, when transistors are fully "ON," depend only on  $R_{\text{DS(ON)}}$ , while at partially opened throttle, when the transistors switch between "ON" and "OFF," additional power losses occur.

The range of  $R_{\text{DS(ON)}}$  lies between 3 and 15 m $\Omega$  and its value is proportional to transistor size. Considering that ESC power losses are a small portion of input power, and the fact that ESC manufacturers do not include in ESC documentation the type of transistors used, we model the ESC as a constant value resistor of  $R_{\text{DS(ON)}} = 5$  m $\Omega$ . BLDC motor ESCs use three pairs of transistors to manage the three phase current, so the total resistance of the ESC will be:

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$$R_{\text{ESC}} = 3R_{\text{DS(ON)}} = 0.015\Omega \quad (9)$$

Another important quantity of ESC is the maximum current  $i_{\text{ESC}}$  they can handle. This appears as a design constraint.

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**Battery Model**

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Due to their high energy density and discharge rate, MRAs use Lithium Polymer (LiPo) batteries. A LiPo pack consists of identical LiPo cells each with a nominal voltage of 3.7 V. Parallel connection of battery packs raises the battery total capacity, while keeping the nominal total voltage the same. Therefore, the nominal total voltage of a LiPo battery is:

$$V_b = n_c 3.7 \tag{10}$$

where  $n_c$  is the number of cells connected in series in a battery pack. The battery has an internal total resistance  $R_{bat,tot}$ . When connected to a load its output voltage is:

$$V_{b,out} = V_b - i R_{bat,tot} \tag{11}$$

where  $i$  is the load current.

Each cell has internal resistance  $R_{sc}$ , capacity  $C_{sc}$ , and maximum discharge rate  $DR_c$ . The total battery capacity is:

$$C_{tot} = n_p C_{sc} \tag{12}$$

where  $n_p$  is the number of battery packs connected in parallel. Each cell's power is:

$$P_{sc} = 3.7 DR_c C_{sc} \tag{13}$$

Each cell's energy is:

$$E_{sc} = 3.7 C_{sc} \tag{14}$$

A battery's total power is:

$$P_{bat,tot} = P_{sc} n_c n_p \tag{15}$$

while its total energy is:

$$E_{bat,tot} = E_{sc} n_c n_p \tag{16}$$

To calculate  $R_{bat,tot}$  we apply Kirchoff's law to a battery consisted of  $n_p$  identical packs connected in parallel, each of which consists of  $n_c$  identical cells connected in series. Each battery pack has an internal resistance:

$$R_i = n_c R_{sc}, \quad i = 1, \dots, n_p \tag{17}$$

The battery total resistance is:

$$R_{bat,tot} = \prod_{j=1}^{n_p} R_j / \sum_{i=1}^{n_p} \left( \frac{1}{R_i} \prod_{j=1}^{n_p} R_j \right) = \frac{(n_c R_{sc})^{n_p}}{n_p (n_c R_{sc})^{n_p-1}} = \frac{n_c R_{sc}}{n_p} \tag{18}$$

**Propeller Model**

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Propellers used on MRAVs are mostly the same propellers used in remote controlled (RC) airplanes. Propeller performance is described by its thrust  $T$ (N), power  $P$ (W), and torque  $M$ (Nm). To model performance in static conditions, we use manufacturer data such as propeller diameter  $D_p$  and its pitch  $p$  at 75 % of its radius. Performance quantities are then related to propeller speed, diameter, and pitch. This is achieved through a number of coefficients.

The thrust coefficient is given by:

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$$C_T = T / \rho (N/60)^2 D^4 \tag{19}$$

where  $T$  is thrust (N),  $\rho$  is air density (kg/m<sup>3</sup>),  $N$  is propeller speed (rpm), and  $D$  is the propeller diameter (m).

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The power coefficient is given by:

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$$C_P = P / \rho (N/60)^3 D^5 \tag{20}$$

where  $P$  is power (W).

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The torque coefficient is given by:

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$$C_M = M / \rho (N/60)^2 D^5 \tag{21}$$

where  $M$  is torque (Nm). Using the fundamental relation between power, torque, and speed we get:

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$$C_M = C_P / 2\pi \tag{22}$$

These coefficients are next related to propeller diameter and pitch. Using the Blade Element Momentum Theory (BEMT) and a series of assumptions [9], we get the following equations for thrust and power coefficients:

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$$C_T = \frac{\pi^3}{4} \frac{1}{2} \sigma C_{l\alpha} \left( \frac{\theta_{0.75}}{3} - \frac{1}{2} \sqrt{\frac{4}{\pi^3} \frac{C_T}{2}} \right) \tag{23}$$

$$C_P = \frac{2}{\pi^2} \frac{C_T^{3/2}}{\sqrt{2}} + \frac{1}{8} \sigma C_{d0} \tag{24}$$

where  $\sigma$  is propeller solidity,  $C_{l\alpha}$  is the slope of blade airfoil lift coefficient–incidence angle curve,  $\theta_{0.75}$  is propeller pitch angle at 75 % of the propeller radius  $R$ , and  $C_{d0}$  is a blade’s airfoil drag coefficient for zero lift.

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To further simplify this model to a restricted propeller size range and geometry, we make the following assumptions. Considering that we refer to geometrically

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scaled propellers, propeller solidity  $\sigma$  will be constant regardless of propeller size. 148  
 Additionally, if the propeller size range is no more than one order of magnitude, 149  
 then the Reynolds number does not change dramatically, so we can assume that the 150  
 aerodynamic quantities  $C_{l\alpha}$  and  $C_{d0}$  are constant. Consequently, thrust and power 151  
 coefficients are only a function of propeller pitch angle  $\theta_{0.75}$ . From the definition of 152  
 geometric pitch we get: 153

$$p = 2\pi R \tan \theta \quad (25)$$

and therefore, the geometric pitch at  $0.75R$  will be: 154

$$p_{0.75} = 2\pi \frac{3}{4} R \tan \theta_{0.75} = \pi \frac{3}{4} D_p \tan \theta_{0.75} \quad (26)$$

Solving Eq. (26) for  $\theta_{0.75}$  we get: 155

$$\theta_{0.75} = \arctan(4/3\pi \cdot p_{0.75}/D_p) \quad (27)$$

Consequently, using Eqs. (23), (24), and (27) we can relate  $C_T$  and  $C_P$  to the ratio 156  
 $p_{0.75}/D_p$  only. Normally,  $\theta_{0.75}$  is in the range of 5–30, resulting a  $p_{0.75}/D_p$  range of 157  
 0.2–1.35. In this region the function  $C_T(p_{0.75}/D_p)$  is linear and this can be shown 158  
 through a numerical solution. Additionally, by observing Eq. (24) we see that  $C_P$  is 159  
 proportional to  $C_T^{3/2}$ , therefore it is proportional to  $(p_{0.75}/D_p)^{3/2}$ , and this can be 160  
 also shown through a numerical solution in the  $p_{0.75}/D_p$  range. 161

Consequently, we get the simplified expressions for thrust and power coefficients: 162

$$C_T = k_1 (p/D_p) + k_2 \quad (28)$$

$$C_P = k_3 (p/D_p)^{3/2} + k_4 \quad (29)$$

where constants  $k_1$  to  $k_4$  can be calculated using experimental data of geometrically 163  
 scaled propellers. 164

Note that to obtain energy efficient propellers at hover, the ratio  $C_T/C_P$  must 165  
 be as high as possible. Solving Eqs. (23) and (24) or (28) and (29), we see that this 166  
 occurs when the ratio  $p/D_p$  is as low as possible, i.e., for a given propeller diameter 167  
 the lowest pitch yields more efficient propellers. 168

## **System Model**

The system model results from the combination of the propulsion system model and 170  
 the equilibrium of forces acting on the vehicle. The propulsion system consists of 171  
 the battery and  $n_{\text{mot}}$  triples of ESC, and of the motors and propellers connected in 172  
 parallel, as shown in Fig. 2. 173

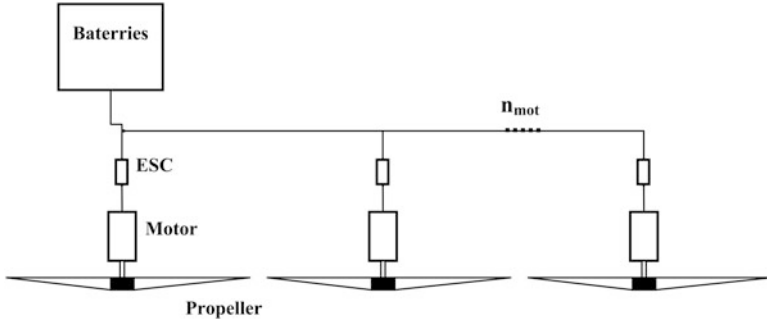


Fig. 2 Propulsion system

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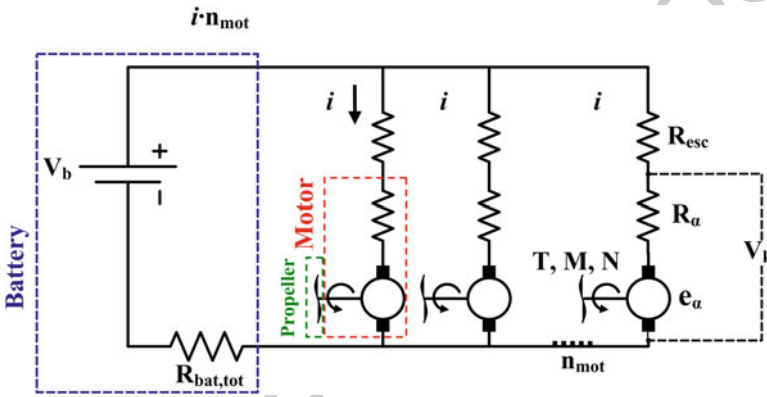


Fig. 3 Propulsion system physical model

The physical model of the propulsion system shown in Fig. 3 combines each component model and outputs the total thrust produced by the  $n_{mot}$  rotors. Assuming that all the rotors have the same speed, the current drawn will be the same for each motor.

Applying Kirchoff's law to the circuit of Fig. 3 we get:

$$V_k + iR_{ESC} = V_b - n_{mot}iR_{bat,tot} \quad (30)$$

$$e_a = V_b - i(R_a + R_{ESC} + n_{mot}R_{bat,tot}) \quad (31)$$

The rotor speed is given by:

$$N = [V_b - i(R_a + R_{ESC} + n_{mot}R_{bat,tot})] K_V \quad (32)$$

The above equation is valid only at full throttle, when the ESC transistors are fully on; otherwise, at partially open throttle, the ESC output voltage is less than the maximum, thus the motor voltage will be less than  $V_k$ .



Equation (32) shows that the motor equivalent resistance is:

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$$R_{\text{tot}} = R_a + R_{\text{ESC}} + n_{\text{mot}} R_{\text{bat,tot}} \quad (33)$$

In this paper we examine the case where the vehicle during a total flight time  $t_{\text{tot}}$  has two operational modes. (a) A *maximum thrust mode* for a percentage  $ATP$  of the total flight time  $t_{\text{tot}}$ , in which motors are at full throttle state producing the maximum static thrust, and (b) a *hover mode*, in which the vehicle hovers for the rest of the flight time. At maximum thrust, the vehicle has the ability to accelerate with an instantaneously maximum acceleration, therefore it has the ability to lift its total weight  $f_w$  times.

**(a) Maximum thrust mode:** The rotor speed is:

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$$N_{\text{acc}} = [V_b - i_{\text{acc}} R_{\text{tot}}] K_V \quad (34)$$

which is equivalent to the following:

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$$N_{\text{acc}} = [V_{k,\text{acc}} - i_{\text{acc}} R_a] K_V \quad (35)$$

where  $V_{k,\text{acc}}$  is the motor supply voltage equal to the maximum ESC output voltage.

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A balance of forces, with a the acceleration, yields:

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$$\begin{aligned} \Sigma F = m_{\text{tot}} a &\Rightarrow n_{\text{mot}} T_{\text{acc}} - m_{\text{tot}} g = m_{\text{tot}} a = (f_w - 1) m_{\text{tot}} g \\ &\Rightarrow n_{\text{mot}} C_T \rho (N_{\text{acc}}/60)^2 D_p^4 = f_w m_{\text{tot}} g \end{aligned} \quad (36)$$

The maximum instantaneous linear acceleration will be:

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$$a = (f_w - 1) g \quad (37)$$

The total mass of the vehicle is:

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$$m_{\text{tot}} = m_{\text{bat,tot}} + (m_{\text{mot}} + m_p + m_{\text{ESC}}) n_{\text{mot}} + m_{\text{frm}} + m_{pl} \quad (38)$$

where  $m_{\text{bat,tot}}$  is the battery total mass,  $m_{\text{mot}}$  is the motor mass,  $m_p$  is the propeller mass,  $m_{\text{ESC}}$  is the ESC mass,  $m_{\text{frm}}$  is the airframe mass, and  $m_{pl}$  is the payload mass.

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The equation of motor-propeller power is:

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$$P_m = P \Rightarrow \{V_b - i_{\text{acc}} R_{\text{tot}}\} (i_{\text{acc}} - i_0) = C_P \rho (N_{\text{acc}}/60)^3 D_p^5 \quad (39)$$

The motor-propeller torque balance yields:

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$$M_m = M \Rightarrow K_T (i_{\text{acc}} - i_0) = C_P \rho (N_{\text{acc}}/60)^2 D_p^5 / 2\pi \quad (40)$$

The system input power is: 203

$$P_{IN,acc} = V_b i_{acc} n_{mot} \quad (41)$$

while the system energy consumption is: 204

$$E_{IN,acc} = P_{IN,tot} ATP \quad (42)$$

**(b) Hover mode:** In this mode, the motor speed is: 205

$$N_{hov} = [V_{k,hov} - i_{hov} R_a] K_V \quad (43)$$

where  $V_{k,hov}$  is ESC output voltage that satisfies  $V_{k,hov} < V_{k,acc}$ . 206

The balance of forces yields: 207

$$\begin{aligned} \Sigma F = 0 &\Rightarrow n_{mot} T_{hov} = m_{tot} g \Rightarrow \\ n_{mot} C_T \rho (N_{hov}/60)^2 D_p^4 &= m_{tot} g \end{aligned} \quad (44)$$

The equation of motor-propeller power is: 208

$$\begin{aligned} P_m = P &\Rightarrow \\ \{V_{k,hov} - i_{hov} R_a\} (i_{hov} - i_0) &= C_P \rho (N_{hov}/60)^3 D_p^5 \end{aligned} \quad (45)$$

while the motor-propeller torque balance gives: 211

$$\begin{aligned} M_m = M &\Rightarrow \\ K_T (i_{hov} - i_0) &= (1/2\pi) C_P \rho (N_{hov}/60)^2 D_p^5 \end{aligned} \quad (46)$$

The system input power is: 214

$$P_{IN,hov} = V_b i_{hov} n_{mot} \quad (47)$$

and the system energy consumption is: 215

$$E_{IN,hov} = P_{IN,hov} t_{tot} (1 - ATP) \quad (48)$$

Battery total power is constrained by: 216

$$P_{IN,acc} \leq P_{bat,tot} \quad (49)$$

while the battery total energy is given by: 217

$$E_{IN,hov} + E_{IN,acc} = E_{tot} = E_{bat,tot} \quad (50)$$

**Parameterization**

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The system equations given in the previous section depend on the functional parameters, which define components performance. Here, these parameters are expressed as a function of component length. This length is taken as the cubic root of a component's volume (cubic length) and is referred to as the *equivalent length*. We do the same with propellers using available experimental measurements. Furthermore, we develop equations that correlate airframe size as a function of propeller diameter, number of rotors, and maximum thrust.

**Electric Motor**

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The electric motors we chose for parameterization are the outrunner BLDC motors from AXI manufacturer. The choice is based on the technical specifications available and on the reliability and performance of these motors.

Here, the equivalent length of each motor is related to the outer dimensions of the motor and not to its stator dimensions. The parameters we want to relate to the equivalent length are the motor armature resistance  $R_a$ , torque constant  $K_T$ , no load current  $i_0$ , and motor mass  $m_{mot}$ . Additionally, motor maximum sustained current (or current capacity)  $i_{max}$  and motor maximum speed  $N_{m,max}$  are parameters that limit motor performance and must be related to equivalent length.

Consequently, we need to develop five equations as functions of equivalent length. After investigation of various correlations of these parameters to the equivalent length, we concluded the following functions due to their optimal fit to manufacturer data. Below,  $R^2$  refers to coefficient of determination, and  $l_{mot}$  to motor equivalent length (m).

$$K_T/R_a = 2.6533 \cdot 10^4 l_{mot}^{3.6032}, R^2 = 0.902 \tag{51}$$

$$K_T^2/R_a = 1.7548 \cdot 10^5 l_{mot}^{5.4833}, R^2 = 0.94 \tag{52}$$

$$M_0 = K_T i_0 = 5.7721 \cdot 10^2 l_{mot}^{3.1888}, R^2 = 0.908 \tag{53}$$

$$M_{max} = K_T (i_{max} - i_0) = 4.5004 \cdot 10^5 l_{mot}^{4.2222}, R^2 = 0.96 \tag{54}$$

$$N_{m,max} = (n_{c,max} 3.7 - i_0 R_a) K_V \Rightarrow$$

$$N_{m,max} = 25604 e^{-17.687 l_{mot}}, R^2 = 0.35 \tag{55}$$

where  $n_{c,max}$  is the maximum number of battery cells in series connection that is proposed by manufacturer.

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To relate motor mass to motor equivalent length, we calculated the mean motor density  $\rho_{\text{mot}}$ : 243  
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$$\rho_{\text{mot}} = 2942 \text{ kg/m}^3 \quad (56)$$

Using (56), the motor mass is: 245

$$m_{\text{mot}} = \rho_{\text{mot}} l_{\text{mot}}^3 \quad (57)$$

### **Electronic Speed Controller** 246

We chose to parameterize ESCs from JETI due to the availability of technical specifications and their performance. Although the ESC is modeled as a constant resistance, additional parameters are needed that relate its operational limit and mass properties to its equivalent length  $l_{\text{ESC}}$  (m). These parameters are the ESC maximum sustained current  $i_{\text{ESC}}$  and ESC mean density  $\rho_{\text{ESC}}$ . 247  
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Using ESC technical specifications, correlations of maximum sustained current  $i_{\text{ESC}}$  and ESC equivalent length  $l_{\text{ESC}}$  are obtained as: 252  
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$$i_{\text{ESC}} = 8.4545 \cdot 10^6 l_{\text{ESC}}^{3.2451}, R^2 = 0.88 \quad (58)$$

The mean ESC density calculated as: 254

$$\rho_{\text{ESC}} = 2580 \text{ kg/m}^3 \quad (59)$$

yielding the ESC mass as: 255

$$m_{\text{ESC}} = \rho_{\text{ESC}} l_{\text{ESC}}^3 \quad (60)$$

### **Battery** 256

We chose to parameterize batteries from Kokam for the same reasons as before. The parameters to be related to battery total equivalent length  $l_{\text{bat}}$  include total power  $P_{\text{bat,tot}}$ , total energy  $E_{\text{bat,tot}}$ , total resistance  $R_{\text{bat,tot}}$ , and mass  $m_{\text{bat}}$ . 257  
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Battery technical specifications concern single battery cells of 3.7 V nominal voltage. However, we need information for any combination of parallel and series connected cells. We assume that  $n_p$  cells connected in parallel result in a larger single cell with volume  $B_{\text{vol}}$ , power  $P_{\text{bat}}$ , energy  $E_{\text{bat}}$ , and internal resistance  $R_{\text{bat}}$ . 260  
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Assuming that the battery consists of  $n_p n_c$  identical cells of volume  $B_{\text{vol,sc}}$  each, then an equivalent battery will consist of  $n_c$  equivalent cells each of which has volume: 264  
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$$B_{\text{vol}} = n_p B_{\text{vol,sc}} \quad (61)$$

Therefore, each equivalent cell volume will be: 267

$$B_{\text{vol}} = l_{\text{bat}}^3 / n_c \quad (62)$$

Applying curve fitting to manufacturer data, the following equation for single cell internal resistance was obtained: 268  
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$$R_{\text{sc}} = 2.84668 \cdot 10^{-7} B_{\text{vol,sc}}^{-0.951154} \quad (63)$$

Correspondingly, the equivalent cell internal resistance is: 270

$$R_{\text{bat}} = 2.84668 \cdot 10^{-7} B_{\text{vol}}^{-0.951154}, R^2 = 0.95 \quad (64)$$

Using (18), (63), and (64), the battery total resistance is: 271

$$R_{\text{bat,tot}} = n_c R_{\text{sc}} / n_p = n_c 2.84668 \cdot 10^{-7} (B_{\text{vol}} / n_p)^{-0.951154} / n_p \Rightarrow \quad (65)$$

$$R_{\text{bat,tot}} = n_c R_{\text{bat}} n_p^{-(1-0.951154)} \approx n_c R_{\text{bat}} n_p^{-0.05} \quad (65)$$

However,  $n_p$  will never be large; therefore using the approximation  $n_p^{0.05}$ , battery total resistance will be: 274  
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$$R_{\text{bat,tot}} = n_c R_{\text{bat}} \quad (66)$$

Applying curve fitting to manufacturer data, we observe that cell energy and power are proportional to its volume. Therefore, using the mean value of the ratios cell energy to cell volume and cell power to cell volume yield: 276  
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$$P_{\text{bat}} = 7.0899 \cdot 10^6 B_{\text{vol}} \quad (67)$$

$$E_{\text{bat}} = 9.0833 \cdot 10^8 B_{\text{vol}} \quad (68)$$

Using (67) and (68), the battery total power and energy are: 279

$$P_{\text{bat,tot}} = n_c P_{\text{bat}} \quad (69)$$

$$E_{\text{bat,tot}} = n_c E_{\text{bat}} \quad (70)$$

The mean battery cell density is calculated as: 280

$$\rho_{\text{bat}} = 1907.8 \text{ kg/m}^3 \quad (71)$$

Yielding the battery total mass: 281

$$m_{\text{bat}} = \rho_{\text{bat}} B_{\text{vol}} n_c \quad (72)$$

**Propeller**

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The propellers we chose to parameterize are taken from APC. The parameters to be related to propeller diameter  $D_p$  and geometric pitch  $p$  are the thrust and power coefficient,  $C_T$  and  $C_P$  respectively.

Previously, it was shown through Eqs. (28) and (29) that for zero flight velocity,  $C_T$  and  $C_P$  are functions of the ratio  $p/D_p$ . The constants  $k_1$  through  $k_4$  in these equations depend on propeller design and the Reynolds number. Here, we are interested in propellers with diameter of 80–500 mm, therefore we use experimental data for these dimensions, so as to satisfy Reynolds number.

Experiments on commercially available propellers used in remote controlled aircrafts were conducted at the University of Illinois, Urbana-Champaign (UIUC) in a wind tunnel [10]. Here, data regarding SPORT type APC propellers are used. From the  $C_T$  and  $C_P$  measurements for these propellers, those that refer to static conditions are used here. We observed that  $C_T$  and  $C_P$  are not affected much by propeller speed; therefore we calculated mean values of  $C_T$  and  $C_P$  for various speeds. These measurements concern propeller diameter of 7 in to 14 in. Finally, the  $C_T$  and  $C_P$  were correlated to the ratio  $p/D_p$ , obtaining the following functions:

$$C_T = 0.0266 (p/D_p) + 0.0793, R^2 = 0.31 \quad (73)$$

$$C_P = 0.0723 (p/D_p)^{3/2} + 0.0213, R^2 = 0.83 \quad (74)$$

The propeller mass is related to propeller diameter  $D_p$  as:

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$$m_p = 0.97573 D_p^{2.5741}, R^2 = 0.98 \quad (75)$$

**Number of Rotors**

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The number of MRV rotors can be even or odd. MRVs with odd number of rotors need an additional degree of freedom (tilting) for one rotor, so that it can vector its thrust and regulate excess torque produced by the rotors. This requires extra mechanisms (revolute joint) and an extra actuator to move the rotor. To take this into account, we assume that these extra mechanisms increase vehicle mass with a percentage  $f_{M,odd}$  of the mass of one of the rods holding the motors. Additionally, actuator power increases total power with a percentage  $f_{P,odd}$  of one motor power. Reasonable values for these coefficients are  $f_{M,odd} = 0.5$  and  $f_{P,odd} = 0.01$ .

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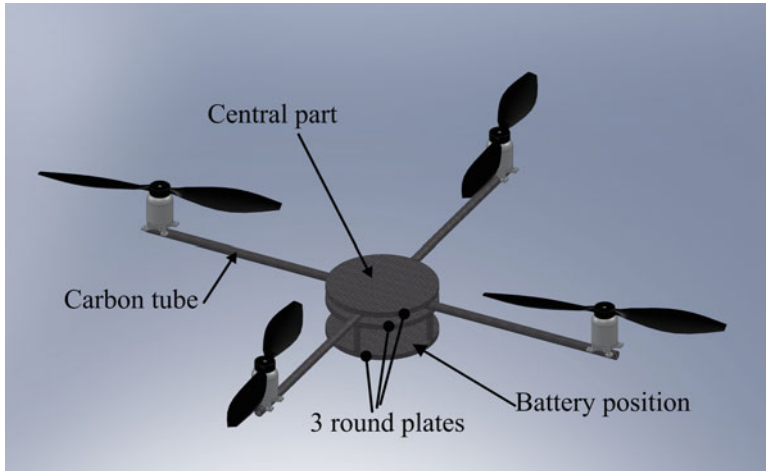


Fig. 4 MRV airframe components

## Airframe

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Here, we are interested in the dimensions and mass properties of an MRV airframe 310  
of simple design, with respect to the number of rotors  $n_{\text{mot}}$ , propeller diameter  $D_p$ , 311  
and airframe loading during flight. 312

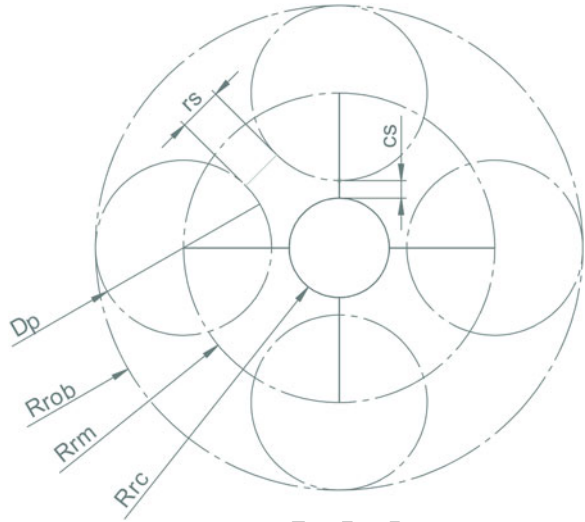
A common rotor configuration is assumed. All rotors are in the same plane and 313  
motors are equidistant lying on a circle with its center coincident to vehicle center. 314  
The number of rotors is in the range of 3–8. 315

To approximate airframe mass its components and material must be assumed. 316  
A reasonable design consists of  $n_{\text{mot}}$  rods to hold the motors, and a central part of 317  
the three circular plates holding the rods and enclosing the battery and electronics. 318  
Additionally, airframe material is carbon fiber due to its high strength to weight 319  
ratio, and the accessories like screws and glue are a percentage  $f_{\text{fr,ac}}$  of each rod 320  
mass. An illustration of such an airframe is presented in Fig. 4. 321

Airframe dimensions are defined by propeller diameter and vehicle loading 322  
during flight. On Fig. 5, airframe dimensions are shown. These include propeller 323  
diameter  $D_p$ , rotor spacing  $r_s$ , central disk-rotor spacing  $c_s$ , center disk radius  $R_{\text{rc}}$ , 324  
motor mounting position radius  $R_{\text{rm}}$ , and radius  $R_{\text{rob}}$  of the circle containing the 325  
whole vehicle. Note that the radius  $R_{\text{rm}}$  is the same for each rod. For a given 326  
propeller diameter, the dimensions  $r_s$  and  $c_s$  define the rest airframe dimensions. 327

The spacing  $r_s$  is important for a number of reasons. Primarily, if  $r_s$  is too 328  
small, there is a danger of adjacent rotor collision during flight due to rod 329  
elasticity. As was shown experimentally in [2] and [6], if  $r_s$  is too small, then 330  
propeller performance deterioration due to adjacent propellers airflow interaction is

Fig. 5 Airframe dimensions



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negligible. Furthermore,  $r_s$  cannot be too small because then the central disk will be very small to accommodate the battery and control unit. Additionally,  $r_s$  cannot be the same for all multi-rotors, i.e., a quadrotor must have a larger  $r_s$  than a hexarotor. For the same reason,  $c_s$  must vary for different number of rotors.

Based on the design trials with respect to the above explanation,  $r_s$  and  $c_s$  were expressed as a function of propeller radius  $R_p$ . Central disk thickness was expressed as a reasonable function of  $R_{rc}$ . For the calculation of carbon tubes' diameter and thickness, we developed equations that take into account material strength, tube maximum deflection, and tube loading. These equations allow calculation of the airframe mass.

## Component Optimal Selection

In the previous sections, component performance was related to component equivalent length. Next, a method is developed for optimal selection of these lengths, which are parameters of the design vector. This vector minimizes an objective function, which is either the vehicle total energy, or the vehicle diameter  $D_{rob}$ .

## Design Parameters

The design requirements are described by a number of parameters set by the designer. These include the payload  $m_{pl}$ , the total flight time  $t_{tot}$ , the payload



capacity described by  $f_w$  indicating how many times the vehicle can lift its own weight, and the factor  $ATP$  which indicates the percentage of total flight time that the vehicle is at maximum thrust mode.

The design vector consists of the number of battery cells  $n_c$  in series, the equivalent battery length  $l_{bat}$ , the equivalent motor length  $l_{mot}$ , the equivalent ESC length  $l_{ESC}$ , the propeller diameter  $D_p$ , the ratio  $p/D_p$ , and the number of rotors  $n_{mot}$ .

### **Design Vector Domain**

The design vector domain results from the size limits of the components that were parameterized earlier. Outside these regions the functions developed earlier may not be valid. Hence, the design vector domain is:

$$0.01 \leq l_{bat} \leq 0.15 \text{ (m)} \quad (76a)$$

$$0.01 \leq l_{mot} \leq 0.08 \text{ (m)} \quad (76b)$$

$$0.005 \leq l_{ESC} \leq 0.05 \text{ (m)} \quad (76c)$$

$$0.05 \leq D_p \leq 0.5 \text{ (m)} \quad (76d)$$

$$0.2 \leq p/D_p \leq 1.5 \text{ (m)} \quad (76e)$$

$$1 \leq n_c \leq 10 \quad (76f)$$

### **Calculation Procedure**

In every optimization step, the requirements vector ( $m_{pl}, t_{tot}, f_w, ATP$ ) is constant, while the design vector ( $n_c, l_{bat}, l_{mot}, l_{ESC}, D_p, p/D_p, n_{mot}$ ) changes until the minimization of objective function is reached.

The calculation procedure follows the following sequence. The battery nominal voltage  $V_b$  is calculated using Eq. (10). Using Eq. (36) we get:

$$N_{acc} = 60 \left( \frac{f_w m_{tot} g}{n_{mot} C_T \rho D_p^4} \right)^{1/2} \quad (77)$$

Using Eq. (40) we get:

$$i_{acc} = K_V C_P \rho \frac{N_{acc}^2}{60^3} D_p^5 + i_0 \quad (78)$$

Using Eq. (35) we get:

$$V_{k,acc} = \frac{N_{acc}}{K_V} + i_{acc} R_a \quad (79)$$

The motor maximum speed without load is:

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$$N_{\max} = [V_b - i_0 R_{\text{tot}}] K_V \quad (80)$$

Using Eq. (41), the maximum total input power  $P_{\text{IN,acc}}$  is calculated, while using Eq. (42) the total input energy at maximum thrust mode  $E_{\text{IN,acc}}$  is calculated. Using Eq. (44) we get:

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$$N_{\text{hov}} = 60 \left( \frac{m_{\text{tot}} g}{n_{\text{mot}} C_T \rho D_p^4} \right)^{1/2} \quad (81)$$

Using Eq. (46) we get:

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$$i_{\text{hov}} = K_V C_P \rho \frac{N_{\text{hov}}^2}{60^3} D_p^5 + i_0 \quad (82)$$

Using Eq. (43) we get:

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$$V_{k,\text{hov}} = \frac{N_{\text{hov}}}{K_V} + i_{\text{hov}} R_a \quad (83)$$

The total input energy at hover  $E_{\text{IN,hov}}$  is obtained using Eq. (48), while using Eq. (50) the total input energy  $E_{\text{tot}}$  is found.

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### **Constraints**

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The constraints result from the independent variable physical consistency. They are given as follows:

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$$\begin{aligned} V_{\text{acc}} - V_b &\leq 0, N_{\text{acc}} - N_{\max} \leq 0, i_{\max} - i_{\text{ESC}} \leq 0 \\ i_{\text{acc}} - i_{\max} &\leq 0, i_{\text{hov}} - i_{\text{acc}} \leq 0, P_{\text{IN,acc}} - P_{\text{bat,tot}} \leq 0 \\ E_{\text{tot}} - E_{\text{bat,tot}} &\leq 0, -i_{\text{acc}} \leq 0, -i_{\max} \leq 0 \end{aligned} \quad (84)$$

### **Optimization Methodology**

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For the calculation procedure, a Matlab program was developed that employs the “fmincon” function (minimum of constrained nonlinear multivariable function) which uses one target deterministic constrained optimization method for nonlinear multivariable objective function.

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Our target was to determine the most energy-efficient design or the smallest one. Hence, the objectives were the minimization of battery energy  $E_{bat,tot}$  or vehicle diameter  $D_{rob}$ , respectively.

In order to check that “fmincon” will not be trapped in local minimums, we also developed a program that scans the whole design vector domain, using nested loops. We observed no differences between these methods after some test runs. Consequently, “fmincon” calculates the total minimum for our objective functions.

**Design Scenarios**

Here we carry out some test runs in order to study the influence of payload and number of rotors on the design vector and the MRV size. In all design scenarios below, the requirement parameters are set to:  $t_{tot} = 15$  min,  $f_w = 2$ ,  $ATP = 0.1$ ,  $f_{fr,ac} = 0.15$ ,  $f_{M,odd} = 0.5$ , and  $f_{P,odd} = 0.01$ . Finally, we compare our program results to commercially available MRVs designs.

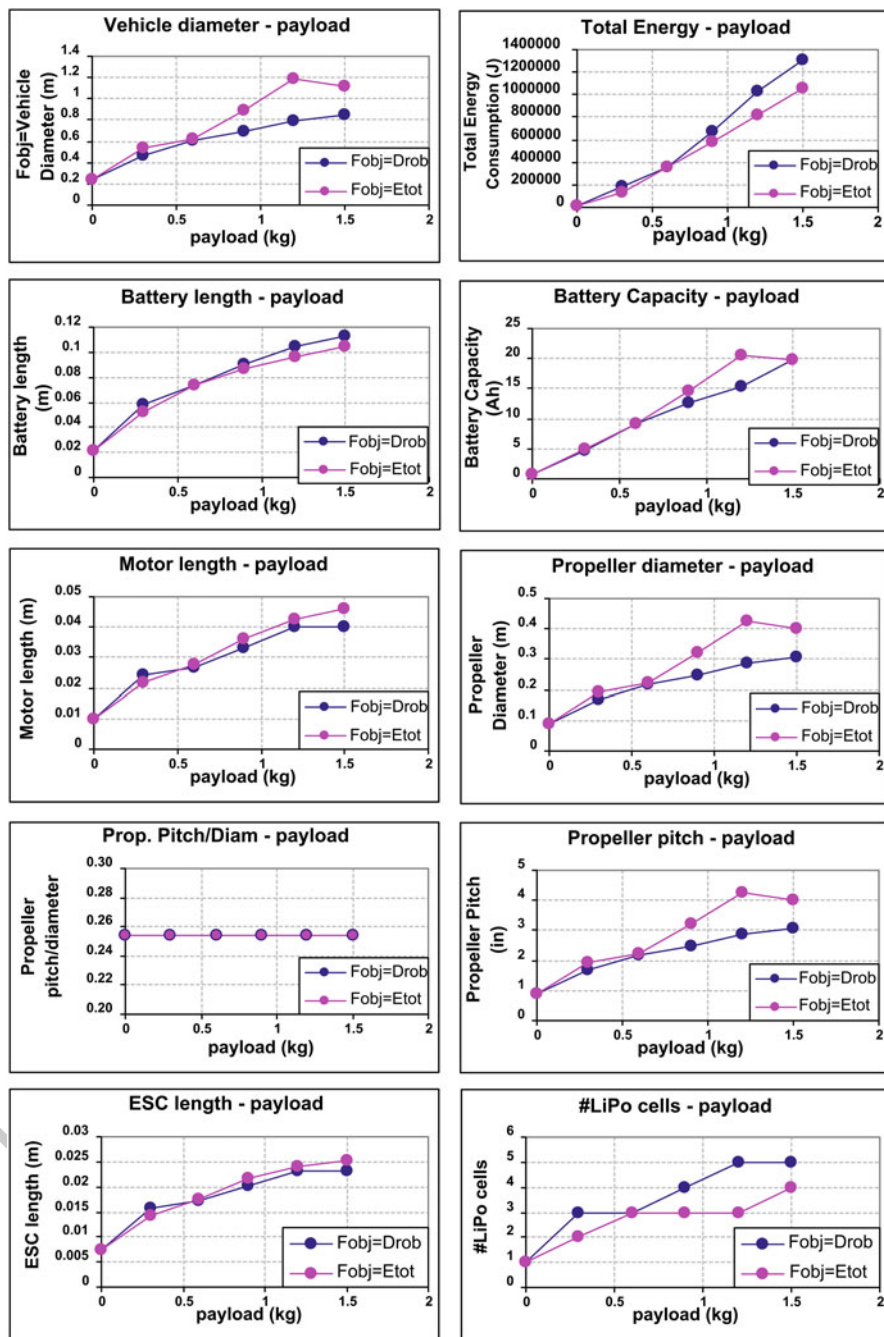
*Study of Parameters Influence*

**Payload Influence**

In this case payload changes from 0 to 1.5 kg, while the number of rotors is constant and equal to 4.

In Fig. 6 the influence of payload on the design vector is shown. In general, we observe that as the payload increases, component equivalent length increases due to power increase. As expected, the ratio  $p/Dp$  is always constant and takes the lowest value permitted, indicating that for a given propeller diameter, the propeller pitch should always be the lowest. In addition, total energy minimization yields a more efficient but a larger design than that obtained by minimizing vehicle size. However, these differences are not large.

In Fig. 7, the influence of payload on total mass and on battery mass is illustrated. We observe that the battery mass is always lower for the minimization of total energy. However, vehicle total mass is not sensitive to the two objective functions. This happens because a smaller vehicle has smaller and therefore lighter motors and rotors. Additionally, observing the battery mass chart, we can say that battery mass increases linearly with payload. For the quadrotor, we can say that we need 1.5 kg batteries for 1 kg payload, and because the flight time is 15 min, then we can say that for 1 kg payload we need 100 g batteries for every minute of flight.



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Fig. 6 Influence of payload on the design vector for the number of rotors equal to 4. Objective functions comparison

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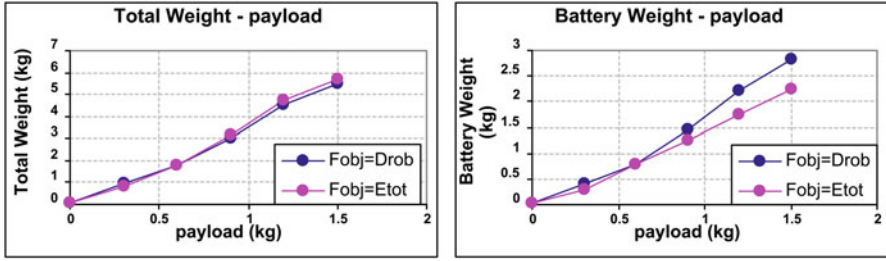


Fig. 7 Influence of payload on the total and battery mass for the number of rotors equal to 4. Objective functions comparison

**Number of Rotors Influence**

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In this case, the number of rotors changes from 3 to 8, while the payload is constant and equal to 1 kg.

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In Fig. 8, the influence of rotors number on the design vector is presented. We observe that for energy minimization, the best design has 8 rotors, but this is true for a payload of 1 kg, see Fig. 9. Additionally, we observe the expected decrease in components equivalent length when the number of rotors increases.

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**Test Cases**

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To determine whether the developed design methodology is valid and yields designs close to reality, we compare program results to two existing commercial MRVs. The first is the quadrotor Walkera HM Hoten X Quadcopter, a small MRV designed for a payload less than 100 g. The other is the Octocopter X88-J2, a large MRV designed for aerial photography and for payloads up to 1.5 kg, see Fig. 10.

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Table 1 presents the quadrotor comparison, with data retrieved from [11]. The payload includes the electronics and control unit. We observe that the program yields results very close to reality. The difference lies on battery configuration and mass. The existing vehicle uses two battery cells in series with total energy 23.7V1Ah=7.4Wh, while the optimized needs 13.7V1.611Ah=6Ah. Therefore, the optimized vehicle seems to be more energy efficient.

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In Table 2 an octorotor comparison is presented, with data taken from [12]. Here we observe that the optimized vehicle is 8 % heavier but 25 % smaller. Also, the optimized vehicle batteries have double capacity because there are two battery cells in series. Thus, the optimized vehicle has total energy 23.7V21.43Ah=159Wh, while the existing vehicle has total energy 43.7V10.6Ah=157Wh. We see that the total energy is almost the same for both the designs.

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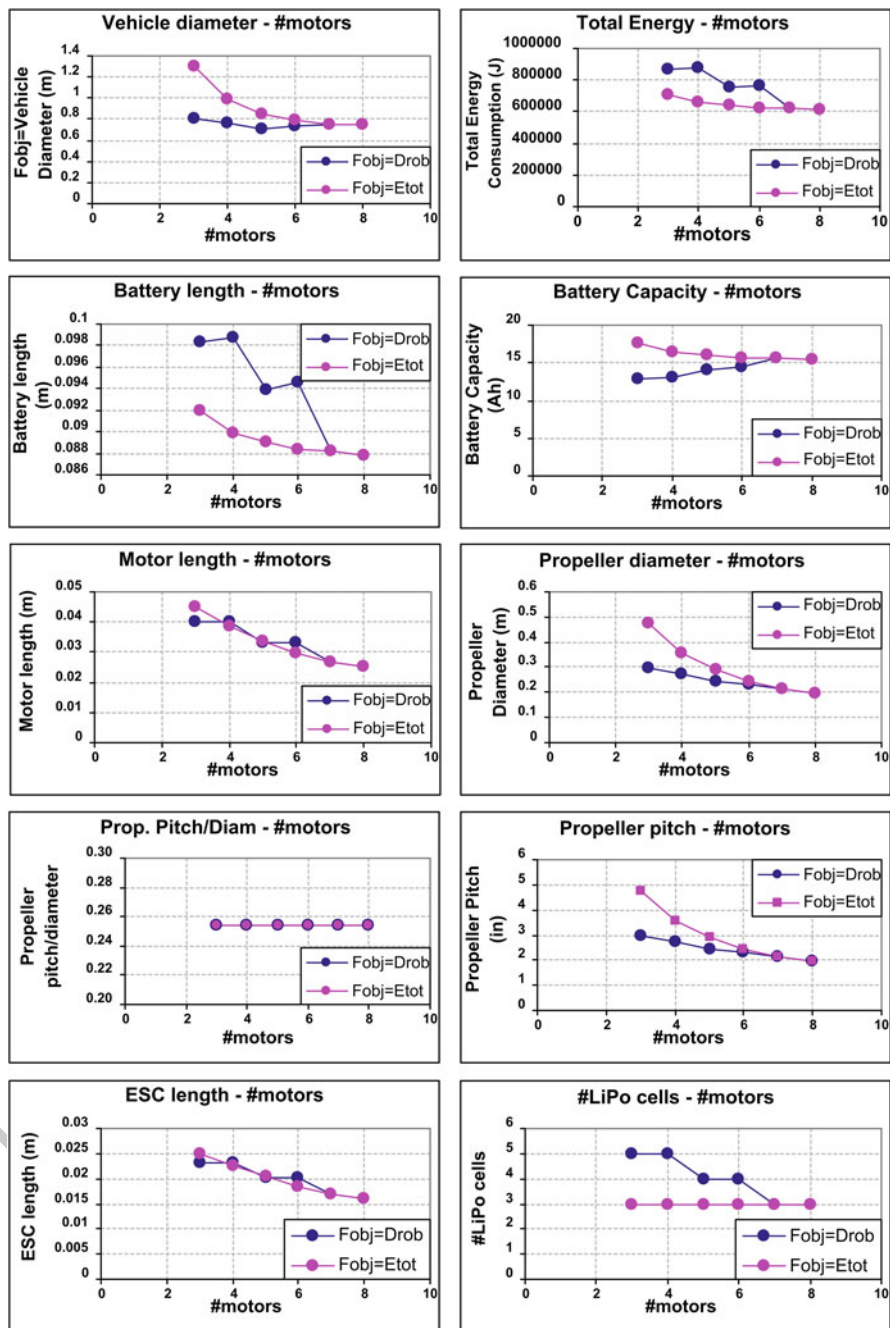
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Fig. 8 Influence of the number of rotors on the design vector for payload equal to 1 kg. Objective functions comparison

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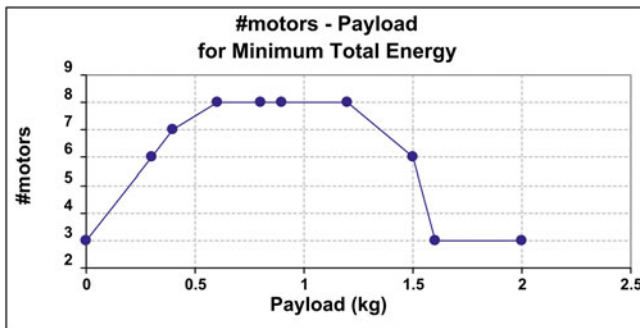


Fig. 9 Influence of payload on the number of rotors for minimum energy

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Fig. 10 (Left) The quadrotor Walkera HM Hoten X Quadcopter. (Right) The Octocopter X88-J2

Table 1 Optimized and actual Walkera HM Hoten X Quadcopter comparison

Model	Walkera Hoten X Quadcopter	Optimization	Difference	
#Motors	4	4	0	t1.1
Payload capacity ( $f_w$ )	2	2	0	t1.2
Total flight time (min)	10	10	0	t1.3
Total mass (kg)	0.332	0.283	-0.05	t1.4
Payload (kg)	0.1	0.100	0.00	t1.5
Vehicle mass (kg)	0.269	0.237	-0.03	t1.6
Battery capacity (Ah)	1	1.611	0.61	t1.7
Battery #cells	2	1	-1	t1.8
Battery mass (kg)	0.064	0.046	-0.02	t1.9
Propeller diameter (m)	0.186	0.184	0.00	t1.10
Vehicle diameter (m)	0.500	0.510	0.01	t1.11
				t1.12

## Conclusions

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This work focused on the parametric design and optimization of a multi-rotor aerial vehicle (MRV). Using simplified models of propulsion system components such as motors, propellers, electronic speed controllers (ESC), and battery, a

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**Table 2** Optimized and actual Octocopter X88-J2 comparison

Model	X88-J2 Octocopter	Optimization	Difference	
#Motors	8	8	0	t2.1
Payload capacity ( $f_w$ )	1.51	1.51	0	t2.2
Total flight time (min)	17.5	17.5	0	t2.3
Total mass (kg)	3.11	3.23	0.12	t2.4
Payload (kg)	1.13	1.13	0.00	t2.5
Vehicle mass (kg)	2	2.10	0.10	t2.6
Battery capacity (Ah)	10.6	21.43	10.83	t2.7
Battery #cells	4	2	-2	t2.8
Battery mass (kg)	1.11	1.22	0.11	t2.9
Propeller diameter (m)	0.305	0.24	-0.07	t2.10
Vehicle diameter (m)	1.205	0.91	-0.29	t2.11
				t2.12

total model for an MRV was created and the whole system performance at hovering and at maximum thrust was described. Additionally, based on the technical specifications of commercially available batteries, motors, and ESCs, component functional parameters were expressed as a function of component size, in terms of an equivalent length. As a result, we were able to calculate system performance as a function of a design vector which consists of each individual component equivalent length. A Matlab program was developed which calculates the optimal design vector using the “fmincon” function. The total energy consumption and the vehicle diameter were considered as objective functions. As a result, for a given payload, payload capacity, number of rotors, and flight duration, the optimal size of each component that minimizes energy or MRV size was calculated. Finally, using the developed program, we were able to study the influence of the payload, and of the number of rotors, on the design vector and the MRV size. The results obtained by the program were compared to existing commercial MRVs, showing that the developed methodology yields designs close to reality. In addition, this methodology provides an MRV designer with the tools to improve an existing design.

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UNCORRECTED PROOF

AUTHOR QUERIES

- AQ1. Please check the sentence “with total energy  $23.7\text{V}1\text{Ah}=7.4\text{Wh}$ , while the optimized needs  $13.7\text{V}1.611\text{Ah}=6\text{Ah}$ ” is ok.
- AQ2. Kindly check the sentence “Thus, the optimized vehicle has total energy  $23.7\text{V}21.43\text{Ah}=159\text{Wh}$ , while the existing vehicle has total energy  $43.7\text{V}10.6\text{Ah}=157\text{Wh}$ ” is ok.

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